

Checking Your Work: Properly Installed HVAC in High-Performance Homes

Home Efficiency Forum October 18, 2019









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Introduction





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- Improper airflow:
 - Average airflow ~20% below target. Blasnik et al. (1995)
 - Average airflow 14% below design. Proctor (1997)
 - Measured airflow ranging from 130 510 CFM / ton. Parker (1997)
 - 70% of units had airflow < 350 CFM / ton. Neme et al. (1999)
 - Improper airflow in 44% of systems. Mowris et al. (2004)

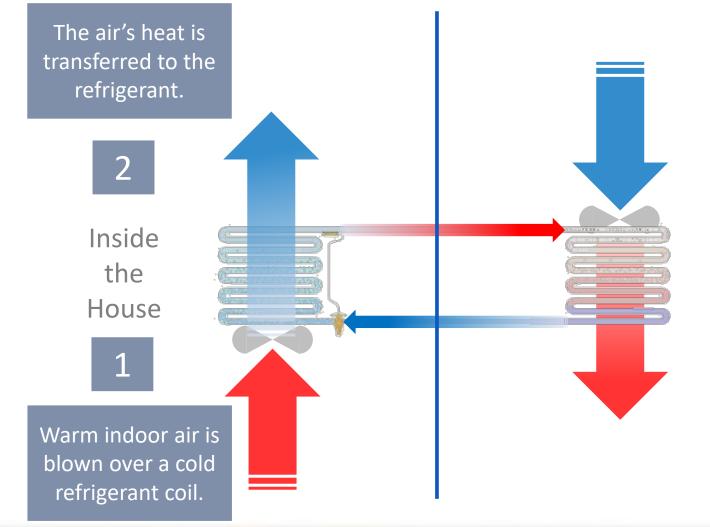


- Incorrect refrigerant charge:
 - In 57% of systems. Downey/Proctor (2002)
 - In 62% of systems. Proctor (2004)
 - In 72% of systems. Mowris et al. (2004)
 - In 82% of systems. Proctor (1997)



Study Author	State	Existing or New Home?	Sample Size	Average Airflow	Airflow <350 cfm	Airflow w/in 10% of 400/ton	Energy Savings	Nata	1							
ology Addio	otate	nomer	0126	Alliow	<220 CIIII	or 400/ton	Potential	Notes					_			
Blasnik et al. 1995a Blasnik et al. 1995b	NV CA	New New	30 10	345 319	50% 90%		8%	Est @ 33% combined char	ge/air fl	low correct	tion ber	efits				
Blasnik et al. 1996	AZ	New	22	344	64%	29%	10%	Eat @ 229/ combined abov								
Hammarlund et al. 1992	CA	New	12		0470	30%	10%	Est @ 33% combined char Single family results	ge/air t	low correc	ction ben	etits				
Hammarlund et al. 1992	CA	New	66		76%	14%	12%	Sincle family results					-			
Neme et al. 1997	MD	New	25	340	10/0	14 70	1270			Existing	a constituted	Charge			Engun	
Palani et al. 1992	n.a.	n.a.	n.a.	540			4%				Samela		N aver	Marialaa	Eneigy	
Parker et al. 1997	FL	Both	27	270	89%	7%	10%	Study Author	Siata			correct to				
Proctor & Pernick 1992	CA	Existing	175	2.0	44%	1 10	1070	Siddy Heleloi	Sidle	Homes?	SIZE	míg spec	charge	charge	Potential	Notes
Proctor 1991	CA	Existing	15			33%		Discribert al 4000-	A 10 /							
Proctor et al. 1995a	CA	Existing	30	300	80%	11%		Blasnik et al. 1995a	NV	New	30	35%	5%	59%	17%	Est @ 67% combined charge/air flow correction benefits
Rodriguez et al. 1995	n.a.	n.a.	n.a.				2%	Blasnik et al. 1995b	CA	New	10				8%	Est @ 67% combined charge/air flow correction benefits
Rodriguez et al. 1995	n.a.	n.a.	n.a.				10%	Blasnik et al. 1996	AZ	New	22	18%	4%	78%	21%	Est @ 67% combined charge/air flow correction benefits
VEIC/PEG 1997	NJ	New	52	372		30%	7%	Farzad & O'Neal 1993	n.a.	n.a.	n.a.					Lab test of TXV; 8% loss @20% overchg; 2% loss @20% underchg
								Farzad & O'Neal 1993	n.a.	n.a.	n.a.				17%	Lab test of Orifice; 13% loss @20% overchg; 21% loss @ 20% underchg
Average				327	70%	22%	8%	Hammarlund et al. 1992	CA	New	12				12%	Single family results
				the monthled the statement				Hammarlund et al. 1992	CA	New	66	31%	61%	8%		Multi-family results
								Ketz 1997	NC/SC	New	22	14%	64%	23%		Charge measured in 22 systems in 13 homes
								Proctor & Pernick 1992	CA	Existing	175	44%	33%	23%		Pasulta from POSE Matul Ensure Comments
								Proctor 1991	CA	Existing	15	44%	00.10	60.0		Results from PG&E Model Energy Communities Program
								Proctor et al. 1995a	CA	Existing	30	11%	3344	EON		Fresno homes
								Proctor et al. 1997a	NJ	New		1170	33%	56%	4.85	
								Rodriguez et al. 1995			52					Est @ 67% combined charge/air flow correction benefits
									n.a.	n.a,	n.a.				5%	Lab test of TXV EER; 5% loss at both 20% overchg & 20% underchg
								Rodriguez et al. 1995	n.a.	n.a.	n.a.				15%	Lab test of Orifice EER; 7% loss @20% overchg, 22% loss @ 20% underchg
								Average				28%	33%	41%	12%	





Outdoor air is blown over the hot refrigerant coil.

3 Outside

Jutside the House



The refrigerant's heat is transferred to the outdoor air.



RESNET/ACCA Std. 310: Guiding Principles

- Take a 'carrot' rather than a 'stick' approach.
- Reward incremental improvement.
- Include procedures applicable to both Rater and HVAC professionals.
- Ensure the procedures provide value in and of themselves.

RESNET/ACCA Std. 310: Grading Concept

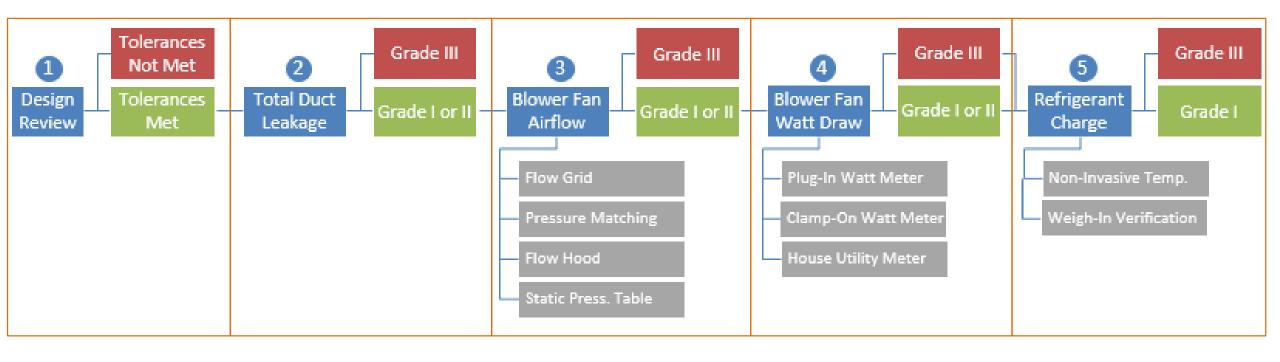
- Follow the insulation quality-installation model:
 - Grade III: The default. No assessment. No penalty and no credit.
 - Grade II: Assessment completed and the system is ok. Partial credit.
 - Grade I: Assessment completed and the system is very good. Full credit.

Overview of Standard 310: Standard for Grading the Installation of HVAC Systems





Std. 310: Standard for Grading the Installation of HVAC Systems





Task 1: Design Review



Task 1: Evaluating the design of the forced-air system

- Rater collects design documentation for the dwelling with the HVAC system under test.
- 2. Rater reviews design documentation for completeness and compares it to the dwelling to be rated. Key features must fall within tolerances defined in the standard. For example:

Floor Area	Outdoor Design Temps	Insulation Levels
Window Area	# Occupants	Infiltration Rate
Indoor Design Temps	Window SHGC	Ventilation Rate

3. If tolerances are met, proceed to next task. Otherwise stop here.





Task 2: Total Duct Leakage



Task 2: Evaluating the total duct leakage

1. Rater measures total duct leakage according to Std. 380, evaluates the results, and assigns a grade:

Grade	Test Stage	# Returns	Total Leakage Limit
I	Rough-In	< 3	4 CFM/100 sqft or 40 CFM
	Rough-In	≥ 3	6 CFM/100 sqft or 60 CFM
	Final	< 3	8 CFM/100 sqft or 80 CFM
	Final	≥ 3	12 CFM/100 sqft or 120 CFM
II	Rough-In	< 3	6 CFM/100 sqft or 60 CFM
	Rough-In	≥ 3	8 CFM/100 sqft or 80 CFM
	Final	< 3	10 CFM/100 sqft or 100 CFM
	Final	≥ 3	14 CFM/100 sqft or 140 CFM
III	N/A	N/A	No Limit

2. If Grade I or II is achieved, proceed to next task. Otherwise stop here.



Task 3: Blower Fan Airflow



- Raters measure the total volumetric airflow going through the blower fan using one of four test methods:
 - A. Flow Grid
 - **B.** OEM Static Pressure Table
 - **C.** Pressure Matching
 - D. Flow Hood
- This is just a single measurement. It is not measuring the airflow from each register and summing those.

Note: Dan rearranged the order to more closely align with NW Rater's current approaches

A. Flow Grid

- 1. Measure static pressure created in supply plenum during operation of HVAC system.
- 2. Install flow grid in filter slot.
- **3**. Measure pressure difference at flow grid and convert to airflow.
- 4. Re-measure static pressure in same location as Step 1, and correct airflow.





A. Flow Grid

Pros	Cons
Easy/simple for many systems	Multiple filter slots in a single system require multiple flow grids
Can work at higher flows	Need to make sure a good seal is achieved around the plate perimeter
	Slightly less accurate +/- 7%
	Requires hole in supply plenum



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B. OEM Static Pressure Table

- 1. Turn on HVAC system.
- 2. Measure external static pressure of system's supply side and return side.
- 3. Determine fan-speed setting through visual inspection.
- 4. Using blower table information, look up total external static pressure and fan-speed setting to determine airflow.

			External Static Pressure, (Inches Water Column)											
MOTOR SPEED	TONS AC ¹	0	.1	0.	.2	0	.3	0	.4	0	.5	0.6	0.7	0.8
SPEED	~~	CFM	RISE	CFM	RISE	CFM	RISE	CFM	RISE	CFM	RISE	CFM	CFM	CFM
High	3	1,498	N/A	1,446	N/A	1,368	N/A	1,302	N/A	1,227	N/A	1,145	1,059	954
Med	2.5	1,223	N/A	1,182	N/A	1,153	30	1,099	31	1,051	32	982	901	813
Med-Lo	2	983	35	971	35	945	36	919	37	878	39	813	746	659
Low	1.5	816	42	794	43	758	45	734	46	678	50	637	597	523



B. OEM Static Pressure Table

Pros	Cons
Inexpensive equipment	Rater required to get OEM Blower Table for installed equipment
Works for systems of all sizes and airflows	Needs carefully-placed hole in supply-side and return-side, sometimes in equipment housing



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C. Pressure Matching



D. Flow Hood





C. Pressure Matching

D. Flow Hood

Pros	Cons	Pros	Cons	
Uses equipment many Raters already own	Can't reach high flows for big systems: needs extrapolation	Accurate: +/- 3%	Can be heavy/unwieldy	
•		Easy to use	Can be sensitive to placement	
Accurate: +/- 3%	Need at least one large return duct or must connect at equipment	Does not require hole in supply plenum	Can be expensive	
	Requires hole in supply plenum		Will not always fit around air inlet	



Task 4: Blower Fan Watt Draw

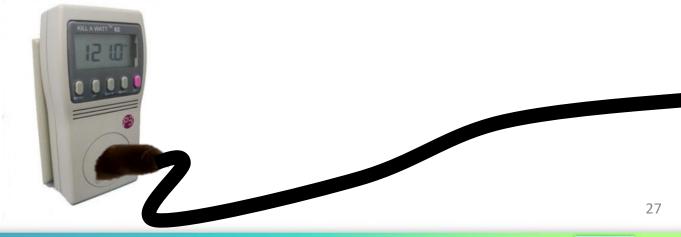


- Raters evaluate the watt draw of the blower fan using one of three test methods:
 - A. Plug-In Watt Meter
 - B. Clamp-On Watt Meter
 - c. Utility Meter



A. Plug-In Watt Meter

- 1. Plug in the watt meter into standard electrical receptacle.
- 2. Plug in the equipment with the blower fan into the watt meter.
- **3.** Turn on equipment in required mode.
- 4. Record reading from portable watt meter.





A. Plug-In Watt Meter

Pros	Cons
Simple	Not usable with hard-wired equipment
Direct measurement of equipment	



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B. Clamp-On Watt Meter

- 1. Turn on equipment in required mode.
- 2. Connect clamp-on watt meter to measure voltage and current at either the service disconnect or through a service panel (not at breaker panel).
- **3.** Record reading from clamp-on watt meter.





B. Clamp-On Watt Meter

Pros	Cons
Useable with hardwired equipment that has service panel or service disconnect	Requires proper training and safety equipment
Direct measurement of equipment	



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C. Utility Meter

- 1. Turn off all circuits except air handler's.
- 2. Turn on equipment in required mode.

For a digital utility meter:

3. Record watt draw from utility meter.

For an analog utility meter:

- 4. For 90+ seconds, record the number of meter revolutions and time.
- 5. Calculate watt draw.







C. Utility Meter

Pros	Cons
Works with all equipment	Indirect measurement, and some meters are less sensitive to low watt draw.
No new equipment needed	Turning off all other circuits can be disruptive

Task 4, the watt draw task, is the only one where <u>you're not required to</u> <u>meet a threshold to proceed</u>. So a Rater could do Task 1-3, skip 4 and take the Grade III default for watt draw, and then proceed to Task 5 and get Grade I for refrigerant charge.





Task 5:Evaluating Refrigerant Charge



Task 5: Evaluating the Refrigerant Charge

- Raters evaluates the refrigerant charge of the system using one of two test methods:
 - A. Non-Invasive Method
 - B. Weigh-In Verification Method Only for select equipment & conditions

Task 5: Evaluating the Refrigerant Charge

A. Non-Invasive Method

- 'Non-invasive' means no gauges connected to refrigerant system.
- Instead, the temperature of the air and refrigerant lines are used.
- Triage systems into two bins:
 - Grade I Charge is okay
 - Grade III Charge is not okay



Refrigerant Gauges Not Connected

Temperature Sensors Used Instead



Task 5: Evaluating the Refrigerant Charge

A. Non-Invasive Method

- 1. Determine SEER and mfr-specified superheat / subcooling value.
- 2. Measure outdoor air and return air temperatures.
- 3. Use to calculate <u>target</u> temperatures for suction line and liquid line.
- 4. Measure <u>actual</u> temperatures for suction line and liquid line.
- 5. Compare <u>target</u> to <u>actual</u> temperatures; if they are close enough, then the system is properly charged.



A. Non-Invasive Method

Pros	Cons
No refrigerant handling certification needed	New procedure to learn
No risk of refrigerant contamination and leaks	Minimum outdoor air temperature limit
Less Rater liability	



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B. Weigh-In Verification Method

- Non-invasive method can't be used for:
 - All outdoor conditions.
 - Mini/multi-split systems.
- In such cases, the weigh-in verification method is used instead.
- Method is primarily a document review rather than a performance test.

B. Weigh-In Verification Method

- Contractor provides:
 - A. Weight of refrigerant added / removed
 - B. Line length and diameter
 - C. Default line length from factory charge (usually 15 feet)
 - D. Factory supplied charge
 - E. Geotagged photo of scale with weight added / removed

- Rater then:
 - 1. Measures line length and diameter
 - Uses lookup table to determine how much refrigerant should have been added / removed
 - 3. Verifies the deviation between the lookup and contractor values are within tolerance
 - 4. Verifies location of geotagged photo matches the location of the equipment



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B. Weigh-In Verification Method

Pros	Cons
No refrigerant handling certification needed	Requires information from contractor
Works at any outdoor temperature	Not a true performance test

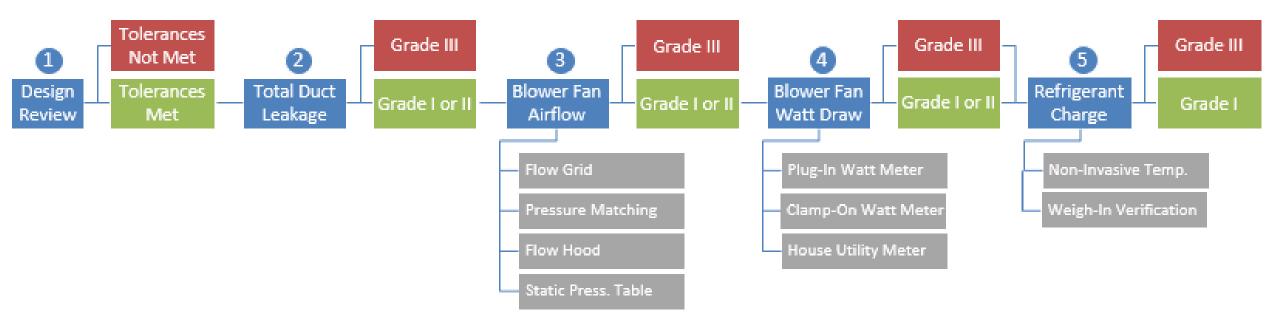


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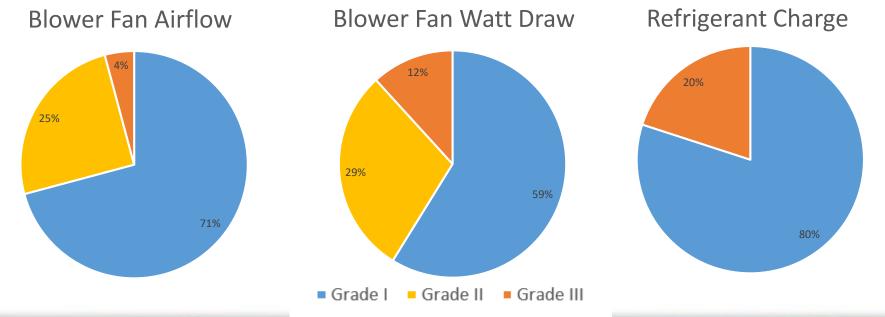


Std. 310: Standard for Grading the Installation of HVAC Systems



Field Test

- Six providers evaluated 18 systems and performed 63 individual tests.
- Required HVAC warm-up time is 15 minutes, but Raters can do other tasks during this time. After that, average time for all tests was **26 minutes**.
- Most systems achieved a Grade I designation:





How HVAC Grading Will Improve Your Homes

#1 - Extra Points in Energy Ratings

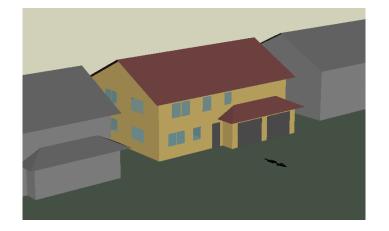
Acknowledgment

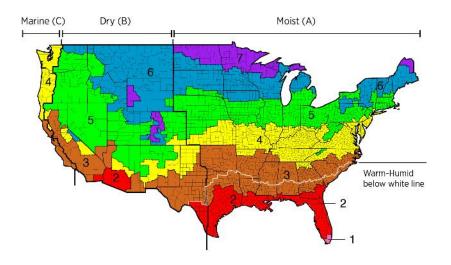
- Jon Winkler, Ph.D.
 - Senior Research Engineer
 - Building Energy Science Group
 - National Renewable Energy Laboratory



House Parameters

- <u>New construction, single-family home</u>
 - 3 bed + 2 bath; 2,500 sq. ft
 - Construction based on 2009 IECC
 - Construction and foundation type varied by climate
 - Simulations followed RESNET Standard 301
- <u>Simulated locations</u>
 - CZ 2 Houston, TX
 - CZ 3 Atlanta, GA
 - CZ 4 Washington, DC
 - CZ 5 Chicago, IL







Equipment Assumptions

- Equipment types
 - SEER 14 air conditioner and gas furnace
 - SEER 14, 8.2 HSPF central heat pump
- Equipment assumptions
 - 0.5 W/cfm fan efficiency
 - Manufacturer recommended airflow is 400 cfm/ton

Defect Scenarios

• Four scenarios were analyzed, where the 'fault' is the % deviation from manufacturer-recommended values:

Parameter		Scenario 2: Airflow Fault		
Airflow defect level	0%	-25%	0%	-25%
Refrig. charge defect level	0%	0%	-25%	-25%

- Generally speaking, in Standard 310:
 - Grade III = -25% fault
 - Grade I = 0% fault



Estimated Maximum ERI Impact

System Type	Location		Max. Point Potential for Going from Scenario 4 (Grade III) to Scenario 1 (Grade I)
	Houston, TX	CZ 2	4.5
AC	Atlanta, GA	CZ 3	2.9
AC	Washington, DC	CZ 4	2.1
	Chicago, IL	CZ 5	0.8
	Houston, TX	CZ 2	6.0
НР	Atlanta, GA	CZ 3	7.0
nr	Washington, DC	CZ 4	6.7
	Chicago, IL	CZ 5	6.1

• Caveats:

- For homes better than 2009 IECC, smaller point potential
- This is the max potential. Many homes will get partial credit.
- Fine-tuning may still occur in Standard 310



Modeling Summary

Previous work by RESNET Working Group:

- Initial estimate of point potential using cursory modeling.
- Air conditioners:
 - Hot climates: ~3 points
 - Mixed climates: ~2 points
 - Cold climates: ~1 point
- Heat pumps: Non-intuitive low potential in cold climates.

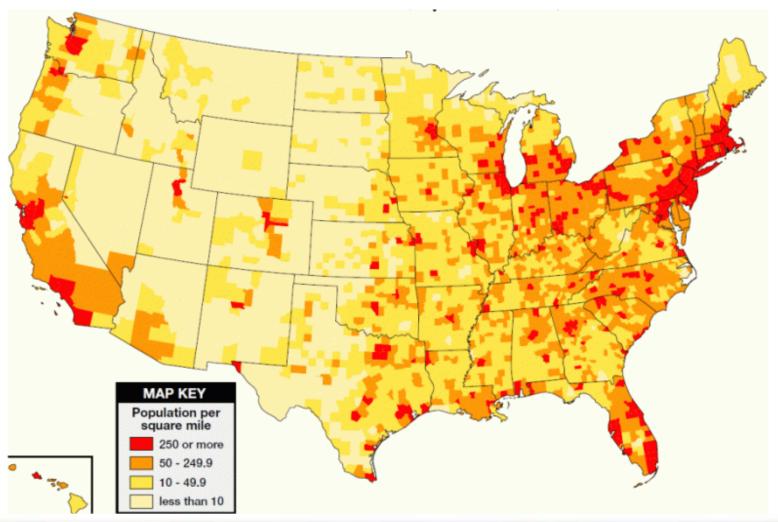
NREL's approach:

- Shows similar trends for air conditioners, but with higher potential, partially due to lower efficiency home.
- More intuitive results for heat pumps.
- Lays groundwork for software programs to ensure installation quality impacts get modeled consistently.



#2 - Provides Alternative to Requirement for Credentialed Contractor

Service providers are harder to find in small markets





HVAC grading provides a new alternative

- You may still choose to work with credentialed contractors.
- But like duct leakage, standard Rater procedures can be used in lieu of a credential.





National Rater Field Checklist ENERGY STAR Certified Homes, Version 3 / 3.1 (Rev. 09)

HVAC System ³⁰ (National HVAC Design Report Item # in parenthesis)					Rater Verified ¹	NA
5. Heating &	Cooling Equi	pment		Correct	venned.	
5.1 HVAC man	ufacturer & me	odel number on installed equipment n	natches either of the following (check box): ²¹			Г
C National	HVAC Design	n Report (4.3, 4.4, & 4.17)	ritten approval received from designer			
			ided test locations and documented below: 32			E
Return-Sic	te External Sta	dic Pressure: WC Sup	ply-Side External Static Pressure:	WC		
			woklist collected, with no items left blank			Γ
			ntilation, Exhaust, & Pressure Balancing Du	cts, Unless No	ted in Foo	inci
			or excessive colled flexible ductwork 21			I.
6.2 Bedrooms	pressure-bala	nced (e.g., using transfer grilles, juny	o ducts, dedicated return ducts, undercut doors) to	_	Г
achieve a	Rater-measur	ed pressure differential 2 -3 Pa and 5	+3 Pa with respect to the main body of the hou alternative compliance option in Footnote 34.			1.1
			connections to trunk ducts, are insulated to 2 R-			t.
			two options. Alternative in Footnote 37: ^{38, 37, 38}			1.5
			A or \$ 40 CFM25, with air handler & all ducts,			Τ
			eddition, all duct boots sealed to finished surface			
Rater-	verified at final	L 30			-	1
			580 CFM25, with the air handler & all ducts, bit			
cavite	a used as duct	ts, duct boots, & register grilles atop t	he finished surface (e.g., drywall, floor) installe	d= 🖬		1
			¹ M25 per 100 sq. ft. of CFA or 5 40 CFM25 ^{30,3}			1
		al Ventilation System				-
		on rate is within either ± 15 CFM or ±1				÷
		fation override control installed and a ne wall switch, but not for a switch th	iso labeled if its function is not obvious (e.g., a	label 🔲		
			vatem, unless controls are installed to operate			╋
			take when not in use (e.g., motorized damper)			1
7.4 System fan	rated ≤ 3 son	es if intermittent and S1 sone if contr	hubus, or exempted as			T
			ICM / ICM (4.7), or the controls will reduce the			Γ,
			te HVAC system is heating or cooling			1.
		GY STAR certified if used as part of				1
7.7 Air inlet location (Complete if ventilation air inlet location was specified (2.12, 2.13); otherwise check 'NA'): ^{40, 47} 7.7.1 Inlet pulls ventilation air directly from outdoors and not from attic, crawlspace, garage, or adjacent dwelling unit						1
			d-string distance from known contamination so		<u> </u>	+
			nd ≥ 3 ft. distance from dryer exhausts and so.			Ι.
	the roof		,		-	
7.7.3 Inlet is	provided with	rodent / insect screen with ≤ 0.5 inch	mesh			Г
8. Local Mech	nanical Exhau		a system is installed that exhausts directly to the		meets one	đ
			inflow and manufacturer-rated sound level stars	dards: ^{63, 68}		
Location		Continuous Rate	Intermittent Rate ²⁵			-
8.1 Kitchen	Airfow	≥ 5 ACH, based on kitchen volume ^{60, 61}	≥ 100 CFM and, if not integrated with range also ≥ 5 ACH based on kitchen volume ⁶⁰	ile 🗖		L
0.1 Michen	Sound	Recommended: \$1 sone	Recommended: 5 3 somes			1
L	Airfow	≥ 20 CFM	2 50 CFM	<u> </u>	-	⊢
8.2 Bathroom	Sound	Required: \$ 1 some	Recommended: 5 3 sones			
9. Filtration						-
9.1 At least on	MERV 6 or h	inher filer installed in each duried m	echanical system in a location that facilitates as			$\mathbf{L}_{\mathbf{z}}$
	ar service by th		,			15
9.2 Filter access panel includes gasket or comparable sealing mechanism and fits anugly against the exposed edge of						E
filter when closed to prevent bycass ⁵⁶ 9.3 All return air and mechanically supplied outdoor air passes through filter prior to conditioning					-	1.
			ough filter prior to conditioning			14
10. Combusti						-
10.1 Furnaces,	boilers, and w	rater heaters located within the home as in Footnote 57, ^{68, 68, 67}	's pressure boundary are mechanically drafted	or 🗆		1
10.2 Fireplaces located within the home's pressure boundary are mechanically drafted or direct-vented. Alternatives in						+-
 Prepades located within the nome s pressure doundary are mechanically draned or direct-vensio. Asematives in Footnote 50, 51, 51, 55 						Ľ
10.3 If unvented combustion appliances other than cooking ranges or overa are located inside the home's pressure					<u> </u>	t
boundary, the Rater has followed Section 802 of RESNET's Standards, encompassing ANSUACCA 12 CH-2014 Appendix A, Section A3 (Carbon Monoxide Test), and verified the equipment meets the limits defined within ^{61,64}						1
	A MERCINA A			_		-
Rater Name: Rater Pre-Drywall Inspection Date: Rater						
Rater Name: Rater Final Inspection Date: Rater						
Builder Employ		Du Du	ider Inspection Date:	Builder Initials		
						_

and t	Natio
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NEWS CONTRACTOR	

onal HVAC Design Report ¹ ERGY STAR Certified Homes, Version 3 / 3.1 (Rev. 09)

HVAC Des

Complete one National HVAC Design Report for each system design for a house plan, created for either the specific plan configuration (i.e., elevation, option, orientation, & county) of the home to be certified or for a plan that is intended to be built with different configurations (i.e., different elevations, options, and/or orientations). Visit <u>www.energyvite.gov/new/sereal/weckelog</u> and see Footnote 2 for more information. • Obtain efficiency features (e.g., window performance, insulation levels, and infiltration rate) from the builder or Home Energy Rater. Provide the completed National HVAC Design Report to the builder or credentialed HVAC contractor and to the Home Energy Rater.

1. Design Overview									
1.1 Designer name:		Der	igner com	pany:				Date:	
1.2 Select which party you are providing these dea	ign service	a to:	Build	er	or		Credential	ed HVAC o	ontractor
1.3 Name of company you are providing these dea	ign service	a to (if diffe	erent than I	tem 1.1):					
1.4 Area that system serves: Whole-house		per-level		wer-level		Other			
1.5 is cooling system for a temporary occupant los	d7 *		Yes	No					
1.6 House plan:		Check box	to indicate	whethert	he system	design is	site-specif	ic or part of	f a group:
Site-specific design. Option(s) & elevation(s)									
Group design. Group #: out of			house pla	n. Con	figuration	modeled:			
2. Whole-House Mechanical Ventilation Desi	gn 4.1								Designs Verified
Airflow:									
2.1 Ventilation airflow design rate & run-t	me meet th	e requirer	nents of At	HRAE 62	2-2010, 2	013, or 20	16 4		
2.2 Ventilation airflow rate required by 62	2 for a con	tinuous sy	atem	CFM					-
2.3 Design for this system: Vent. airflow	ratio:	CFM R	un-time pe	r cyde:	minut	es Cycle	time:	minutes	-
System Type & Controls:									
2.4 Specified system type: Supp	ly 🗖 🗈	shauat	Bala	nced					-
2.5 Specified control location:					(e.c	. Master I	bath, utility	room)	-
2.6 Specified controls allow the system to	operate as	Annatical	e without a	coupart is					
2.7 Specified controls include a readily-ar							d if its fury	dion is not	
obvious (e.g., a label is required for a									
2.8 No outdoor air intakes designed to co	nnect to the	e return six	te of the H	VAC syste	m, uniess	specified	controls op	erate	
intermittently and automatically based	on a timer	and restri	ct intake wi	hen not in	use (e.g.,	motorized	damper) ?		
Sound: 2.9 The fan of the specified system is rate	ed ≤ 3 sone	a if intermi	thent and s	1 sone if	continuous	, or exemp	pted *		
Efficiency:									
2.10 If system utilizes the HVAC fan, then								will reduce	
the standalone ventilation run-time by							ling		
2.11 # betwoom fans are specified as pa									
Air Inlet Location: (Complete this section if syster									
2.12 Inlet pulls ventilation air directly from									
2.13 Inlet is ≥ 2 ft. above grade or roof de							ion source	s (e.g.,	
stack, vent, exhaust, vehicles) not exi	ting the roo	t, and ≥ 3	t, from kno	WIT SOUTCE	a exting t	he roof			
3. Room-by-Room Heating & Cooling Loads							_		
3.1 Room-by-room loads calculated using: Unat			_	-	fRAE Fun	damentais	Dthe	r per AHU 1	· -
3.2 Indoor design temperatures used in loads are									
3.3 Outdoor design temperatures used in loads: (!	See Footno	te 12 and	energyster	govihvaco	lesignlang				-
County & State, or US Territory, selected:	_		0	cooling sea	aon:	*F Hea	ting seaso	n:1	1
3.4 Number of occupants used in loads: 13									-
3.5 Conditioned floor area used in loads:					iq.Pt.				-
3.6 Window area used in loads:					iq. Ft.				-
3.7 Predominant window SHGC used in loads: 15								_	-
3.8 Infiltration rate used in loads: ¹⁶	Summer				Wint	er:			-
3.9 Mechanical ventilation rate used in loads:				(2FM				-
Loads At Design Conditions (kBtuh)	N	NE	E	SE	5	SW	W	NW	-
3.10 Sensible heat gain (By orientation ¹⁴)									-
3.11 Latent heat gain (Not by grientation)						-			-
Cooling 3.12 Total heat gain (By orientation ¹⁴)									-
3.13 Maximum - minimum total heat gain	(tem 3.12)	across or	entations -		kBluh	Variate	on is ≤6 ki	Stuh 14, 17	
				the second se					
Heating 3.14 Total heat loss (Not by orientation)									

Revised 09/01/2018

National HVAC Commissioning Checklist 1, 2

	ENERGY STAR Certified Hor	nes Version 3/31(Rev 0	9)			
The commissioning contactor must be credentialed by an HVAC oversight organization to complete the shackstaf. One checklast must be completed checkstaf for each HVAC heavy with the corresponding National HVAC bears that be networked by the contactor for a minimum of there years for quildy assurance purposes. Furthermore, the contractor shall provide the completed checkstat to the builder, the Home Energy Ruler responsible for cartifying the home, and the HVAC oversight organization upon request. Vita twee aneropeate contractor parts of information about the credential requirement and this checkstat. Contractor name Contractor name Contractor company is conductiated with: CACCA Advanced Energy INSERDA L3 Departation they our company is conductiated with: CACCA Advanced Energy NYSERDA L3 Departation they our company is conductiated with: CACCA Advanced Energy NYSERDA L3 Abone Advanced Energy NYSERDA L3 Abone Advanced Energy NYSERDA L4 Home addeesa: CDF State: CDF State: CDF Company (CDF State) Contractor write the state of the conduction of the context of the c			101.0	57			
The completed checklish for each commissioning contractor for seeh HVAC system That is commissioned. The completed checklish to be builder, the hore server for quality assumes purposes. Furthermore, the contractor and provide the completed checklish to the builder, the hore Energy Rear responsible for curritying the hores, and the HVAC oversight or generation upon request. Vest even summarizes and control to the system should the credential requirement and this checklist. Commissioning Overview Contractor company about the control of the system has been collected from designer or builder. Lo Contractor company is credentialed with: Contractor company. Date		inter examination to complete this should be	One should	-			
The completed checklist for each commanisationed system, along with the corresponding National HVAC Design Paperds at half be instanced by the contractor shall provide the completed checklist to the builder, the Home Energy Ruler responsible for cartifying the home, and the HVAC oversight organization upon request. Vest <u>wave accesses to contractor shall provide the contractor shall provide the completed checklist. Contractor name. Contractor name. Contractor company is credential event in the checklist. Contractor name. Contractor name. Contractor company is credential event in the checklist. Contractor name. Contractor name. Contractor name. Contractor name. Contractor company is credential event in the checklist. Contractor name. Con</u>			C One check				
the completed checklast to the builder, the Home Energy Rater responsible for certifying the home, and the HVAC oversight organization upon request. • Vait wave supervised conventioned on the state of the condential requirement and this checklast. • Commissioning Overview • Contractor same • All other dimentiated • All other dimentiates • Contractor vertified • Contractor vertified <td></td> <td></td> <td>port, shall be</td> <td></td>			port, shall be				
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View queue senservalue convinced homestings: for information about the credential requirement and this checklat. Commissioning Overview Comparisation that your company is credentialed with: ChACA Advanced Energy NYSERDA Subice clines and an advanced Energy NYSERDA Subice clines and NYAC Design Report: Company is credentialed with: Chip Subice clines (Chappe - Report) Subice (Chappe - Report)		ible for certifying the home, and the HVAC	cversight				
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1.2 Organization that your company is credentiated with: IAACCA IAdvanced Energy INYSERDA 1.3 Builder client name: IAAccA State 2p code: 1.4 Home addware: Chr. State 2p code: 1.5 National HVAC Design Report corresponding to this system has been collected from designer or builder. IConstructor-verified 1.6 Area that system serves, per Ben 1.6 of National HVAC Design Report: IS the appetific design ICoroup design ICorope design ICoroup design ICoroup d			1	_			
1.3 Bulder client name: Chp State Zp code: 1.4 Home address: Chp State Zp code: 1.5 National HVAC Design Report corresponding to this system tables. Clips: State: Zp code: 1.7 House plan, per lien 1.8 of National HVAC Design Report: Bite-specific design: Contractor-verified 1.7 House plan, per lien 1.8 of National HVAC Design Report: Bite-specific design: Contractor-verified 1.7 Motes plan, per lien 1.8 of National HVAC Design Report: Bite-specific design: Contractor-verified 2.1 Outdoor ambient temperature states that the system state in table science. 7 OB - 2.1 Outdoor ambient temperature states that the states . 7 OB - 2.3 Liquid Ine temperature: 7 OB - - 2.4 Liquid Ine temperature: 7 OB - - 2.5 Suction line temperature: 7 OB - - 2.5 Suction line temperature: 7 OB - - 2.5 Suction line temperature: 7 OB (Uarg Item 2.3) - - - 2.6 Suction line temperature: 7 OB (Uarg Item 2.3) - - - - 2.6 Suction line temperature:				-			
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1.5 National HVAC Design Report corresponding to this system has been collected from designer or builder. Contractor-verified 1.6 Area that system satures, per item 1.4 of National HVAC Design Report. Consup design # 2.1 House plane, per item 1.6 of National HVAC Design Report. Consup design # 2.1 House plane, per item 1.6 of National HVAC Design Report. Consup design # 2.1 Outdoor ambient temperatures at condurates. TOB 2.1 Outdoor ambient temperatures at condurates. TOB 2.1 Outdoor ambient temperatures at condurates. TOB 2.3 Lipstd lines pressure: PBB 2.4 Lipstd lines pressure: P1 DB 2.5 Suction line temperature. TOB (Using Item 2.1) 2.6 Suction line temperature: TOB (Using Item 2.3) 2.7 Ordinares authantion temperature: TOB (Using Item 2.5) 2.7 Ordinares authantion temperature: TOB (Using Item 2.5) 2.7 Ordinares authantion temperature: TOB (Using Item 2.5) 2.7 DOB subcooling geal: TOB (Using Item 2.5) 2.8 Subcooling value: TOB (Using Item 2.5) 2.9 OEM subcooling geal: TOB (Using Item 2.5) 2.1 Souther at the subsect and to subsecoling or subsect and to subsect and to subsect and to s							
1.6 Area that system serves, per lism 1.4 of National HVAC Design Report: Site-specific design _ Croup design # 1.7 House pins, per lism 1.8 of National HVAC Design Report: Site-specific design _ Croup design # 2.8 Refrigerent Charge - Ren using the 1.5 of National HVAC Design Report: Site-specific design _ Croup design # 2.1 October ampartures using the 1.5 of National HVAC Design Report: Site specific design _ Croup design # 2.1 October ambient temperatures and conducts seture design in the 2.1, and the contracts seture design in the 1.5 of National HVAC Design Report: Y B 2.1 October ambient temperature in alid duct near exponentor, during cooling mode: Y WB - 2.1 October ambient temperature: Y DB - - 2.2 Return-side air temperature: Y DB - - 2.3 Lipid line temperature: Y DB - - 2.5 Suction line pressure: Y DB - - 2.6 Suction line pressure: Y DB - - - 2.6 Suction line pressure: Y DB - - - - 2.6 Suction line pressure: Y DB - - - - - - - - - - - - - <				-			
1.7 House plan, per Hem 1.6 of National HVAC Design Report: <pre></pre>				rified			
Procession Contractor Contractor Contractor Procession Environs. Tokewise them multicher environment operating impresentate or the conding cycles. Ben the system shall be holded Contractor 2.10 Outfoor ambient temperature at condenses. "7 D8 - 2.21 Neumoids at temperature inside duct near evaporator, during cooling mode: "7 D8 - 2.3 Liquid time pressure: "7 D8 - - 2.4 Liquid time pressure: "7 D8 - - 2.5 Suction line temperature inde duct near evaporator, during cooling mode: "7 D8 - - 2.5 Suction line temperature: "7 D8 - - - 2.6 Suction line temperature: "7 D8 - <td< td=""><td>1.6 Area that system serves, per item 1.4 of National HVAC Design Report: W</td><td>hole-house Upper-level 🗌 Lower-level 🔲 🕻</td><td>Other</td><td></td></td<>	1.6 Area that system serves, per item 1.4 of National HVAC Design Report: W	hole-house Upper-level 🗌 Lower-level 🔲 🕻	Other				
i known, below the maintacture recommended minimum operating temperature for the cooling cycle, then the system studie include is a traffield NA in the X-1, and the contractor studie deck: NA' in this Section." 2.1 Outdoor ambient temperature at conductors: 2.2 Return-side at temperature inside duct mar evaporator, during cooling mode: 7 WB 2.1 Outdoor ambient temperature inside duct mar evaporator, during cooling mode: 7 WB 2.2 Return-side at temperature inside duct mar evaporator, during cooling mode: 7 WB 2.2 Return-side at temperature: 7 DB 2.3 Ligitable ine pressure: 7 DB 2.4 Ligitable ine pressure: 7 DB 7 DB 7 Dg tem vite imperature inside duct mar evaporator, during cooling mode: 7 VB 2.5 Suction line pressure: 7 DB 2.6 Suction line temperature: 7 DB 2.7 Condenser saturation temperature: 7 DB 2.7 Condenser saturation temperature: 7 DB 2.1 DB (bleng tem 2.3) 2.2 Return vite imperature: 7 DB (bleng tem 2.3) 2.2 Subcooling deviation: 7 DB (bleng tem 2.5) 2.1 Subcooling deviation: 7 DB (bleng tem 2.11) 2.1 Superheat deviation: 7 DB (bleng tem 2.12) 2.1 Superheat deviation: 7 DB (bleng tem 2.12) 2.1 Superheat deviation: 7 DB (bleng tem 2.12) 2.1 Superheat deviation: 7 DB (bleng tem 2.12 - blem 2.13) 2.1 Subcooling deviation: 7 DB (bleng tem 2.12 - blem 2.14) 2.2 Subcooling deviation: 7 DB (bleng tem 2.12 - blem 2.14) 2.3 Subcooling deviation: 7 DB (bleng tem 2.12 - blem 2.14) 2.3 Subcooling deviation: 7 DB (bleng tem 2.12 - blem 2.14) 2.3 Subcooling deviation: 7 DB (bleng tem 2.12 - blem 2.14) 2.4 An CBM tem 2.14 is a ST 2.3 MA nCBM tem 2.14 is a ST 2.3 Mancend text estamel static pressure: 7 DB (bleng tem 2.12 - blem 2.14) 3.3 Massure deviation: 7 DB (bleng tem 2.12 - blem 2.14) 3.3 M	1.7 House plan, per Item 1.6 of National HVAC Design Report:	Site-specific design 🖾 Group de	mign #:				
a TXV, the oxtition temperature at condense:			Contractor				
2.1 Outdoor ambient temperature at condense: "7 DB - 2.2 Return-side air temperature inside duct near exponsion, during cooling mode: "7 WB - 2.3 Liquid time temperature: "7 DB - - 2.4 Liquid time temperature: "7 DB - - 2.5 Subcin time pressure: "7 DB - - 2.5 Subcin time pressure: "7 DB - - 2.6 Subcin time temperature: "7 DB - - 2.7 Ordenser saturation temperature: "7 DB - - 2.8 Subcooling quale: "7 DB - - - 2.10 Subcooling quale: "7 DB 1 DB - - - 2.10 Subcooling deviation: "7 DB (Lining Item 2.5) - <td< td=""><td></td><td></td><td>Verified</td><td>NA</td></td<>			Verified	NA			
2.2 Reham-aide air temperature inside duct near evaporator, during cooling mode: 7 WB . 2.3 Ligidal line pressure:							
2.1 Ligad line pressure: psig - 2.4 Ligad line temperature: "T DB - 2.5 Suction line temperature: "T DB - 2.6 Suction line temperature: "T DB - 2.7 Condense subarition temperature: "T DB - 2.7 Condense subarition temperature: "T DB - 2.8 Succoing value: "T DB - 2.9 ORM subcooling value: "T DB - 2.9 ORM subcooling value: "T DB - 2.9 ORM subcooling value: "T DB - 2.10 Subcooling value: "T DB (Ling spartheat bales and hema 2.1) 2.10 Subcooling value: "T DB (Ling spartheat bales and hema 2.1) - 2.11 Supper-heat deviation: "T DB (Ling spartheat bales and hema 2.1 & 2.2) - 2.13 DEM super-heat goodes and deviation: "T DB - - - 2.13 Subscondup deviation: "T DB Ling spartheat deviation: - -		Contraction of the second s		-			
2.4 Liquid line temperature: "7 DB . 2.5 Suction line pressure: psig . . 2.6 Suction line temperature: "7 DB . . 2.6 Suction line temperature: "7 DB . . 2.7 Orderiser saturation temperature: "7 DB . . . 2.8 Subcooling yake: "7 DB .			-				
2.5 Suction line pressure:							
2.6 Suction line temperature: 'P DB - For Syntem with Thermal Expansion Valve (TXV): - - 2.7 Condense subaration temperature: 'P DB (Using Item 2.3) - - 2.8 Subcooling value: 'P DB (Item 2.7 - Item 2.4) - - - 2.9 ORM subcooling genit: 'P DB (Item 2.6 - Item 2.4) -			-				
For System with Thermal Expansion Valve (TXV): - 2.7 Condenses subarison temperature: T DB (Using Item 2.3) - 2.8 Subcooling goal: T DB (Barn 2.7 - Item 2.4) - 2.9 OEM subcooling goal: T DB (Barn 2.7 - Item 2.4) - 2.10 Subcooling deviation: T DB (Barn 2.8 - Item 2.9) - Der System with Thad Office: - - 2.11 Exaporator saturation temperature: T DB (Barn 2.6 - Item 2.11) - 2.12 Superheat value: T DB (Barn 2.6 - Item 2.13) - 2.13 OEM superheat goal: T DB (Barn 2.6 - Item 2.13) - 2.14 Superheat deviation: T DB (Barn 2.6 - Item 2.13) - 2.15 Item 2.10 is a 3TF or Item 2.14 is a 5TF - - 2.16 An OEM Item VACE Fan Arithew - - - 3.1 The mode with the higher design HVAC fan aiflow used, per Item 5.2 of National HVAC Design Report: - - 3.1 Blace DFMACE Fan Arithew - - - - 3.2 Static pressure leaf heles notion for supply external static pressure: Plenum : Cabinet Transition Other: - - 3.1 Mose HAVAC Fan Arithew - - - - 3.2 Stat			-				
2.7 Condenser saturation temperature: T DB (Using Item 2.3) - 2.8 Subcooling value: T DB (Item 2.7 - Item 2.4) - 2.9 OEM subcooling geal: T DB (Item 2.7 - Item 2.4) - 2.10 Subcooling deviation: 'F DB (Item 2.7 - Item 2.4) - 2.10 Subcooling deviation: 'F DB (Item 2.8 - Item 2.9) - 7 DB (Using Item 2.5) - - 2.11 Evaporator saturation temperature: 'F DB (Using Item 2.5) - 2.12 Superheat value: 'F DB (Item 2.6 - Item 2.13) - 2.13 Subcooling deviation: 'F DB (Item 2.12 - Item 2.13) - 2.14 Superheat value: 'F DB (Item 2.12 - Item 2.13) - 2.15 Item 2.10 its a 3F or Item 3.14 its a 5FF - - 2.16 Item 2.00 its a 3F or Item 3.14 its a 5FF - - 2.16 Item 2.00 counsentation has been attached that defans this procedure - - 3.1 The mode with the higher design MVAC fan attRow used, per Item 5.2 of National MVAC Design Report: - - 3.1 The mode with the higher design MVAC fan attRow used, per Item 5.2 of National HVAC Design Report: - - 3.1 The mode with the higher design MVAC fan attRow used, per Item 1.5 and Na spaceasete. - -		*F DB	-				
2.8 Subcooling value: T DB (Item 2.7 - Item 2.4) . 2.9 OtM subcooling genit: T DB . 2.9 OtM subcooling genit: T DB (Item 2.8 - Item 2.9) . 2.10 Subcooling deviation: T DB (Item 2.8 - Item 2.9) . 7 DB (Item 2.8 - Item 2.9) . . 7 DB (Item 2.8 - Item 2.9) . . 7 DB (Item 2.8 - Item 2.9) . . 2.13 Subprived values: T DB (Item 2.6 - Item 2.1) . 2.13 OtM superheat value: T DB (Item 2.6 - Item 2.1) . 2.14 Superheat deviation: T DB (Item 2.6 - Item 2.13) . 2.15 Item 2.10 is 3 2F or Item 2.14 is 3 5F . . 2.16 An OEM Itest procedure (e.g., as defined for a ground-source heat pump) has been used in place of the sub-cooling or pump-heat process and documentation has been stack-out that definate this procedure . 3.1 Indeer MVAC Fan Arifleer . . . 3.2 Static pressure test helps the design HVAC fan ariflow used, per Item 5 2 of National HVAC Design Report: . . 3.4 Measured returns esternal static pressure: PlenumCobinst [Transition Other: . . 3.4 Measured static pressure: PlenumCobinst [Transition Other: <td< td=""><td>For System with Thermal Expansion Valve (TXV):</td><td></td><td></td><td></td></td<>	For System with Thermal Expansion Valve (TXV):						
2.9 OEM subcooling geal: T DB -	2.7 Condenser saturation temperature: "# DB (Using Item 2.3)	1	-				
2.10 Subcooling deviation: 'T DB (Item 2.6 - Item 2.9) - For System with Fixed Orifice: 'T DB (Using Item 2.5) - 2.11 Exoporter saturation temperature: 'T DB (Using Item 2.5) - 2.12 Superheat value: 'T DB (Using Item 2.5) - 2.13 Superheat value: 'T DB (Using Item 2.5) - 2.14 Superheat deviation: 'T DB (Using Item 2.14) - 2.15 Stars 2.10 Ite 3 3'F or Item 2.14 ite 3.5'F - - 2.15 Item 2.10 Ite 3 3'F or Item 2.14 ite 3.5'F - - 2.15 Item 2.10 Ite 3 3'F or Item 2.14 ite 3.5'F - - 2.16 An DEM test process and down items 'T DB (Item 2.16 - Item 2.16) - 3.16 mooth test process and down items - - - 3.17 he mooth with the higher deving MVAC fram atthow used, par Item 5.2 of National MVAC Design Report: - - 3.16 mooth the supple weigh MVAC fram atthow used, par Item 5.2 of National MVAC Design Report: - - 3.17 he mooth with the higher deving MVAC fram atthow used, par Item 5.2 of National MVAC Design Report: - - 3.17 Ham mode tastice pressure: Plenum Cabriest: - - 3.2 Stafic possure test holes location for supply rederma	2.8 Subcooling value: "# DB (Item 2.7 - Item)	2.4)	-				
Por System with Fload Onfice: If DB (Using Isen 2.5) - 2.11 Exaporator saturation temperature: If DB (Using Isen 2.5) - 2.13 Sophmat value: If DB (Using Isen 2.5) - 2.13 Sophmat value: If DB (Using Isen 2.5) - 2.13 OEM superheat goal: If DB (Using superheat tables and Items 2.1 & 2.2) - 2.14 Superheat deviation: If DB (Isen 2.6 - Isen 2.13) - 2.15 Isen 2.10 is a 37F of Item 2.14 is a 57F - - 2.16 An OEM Iset proceedure (e.g., as defined for a ground-source heat pump) has been used in place of the sub-cooling or more-heat proceedure (e.g., as defined for a subc-real that definant this procedure - 3.1 Indoor HVACF Fan ArtHoe - - - 3.1 Indoor HVACF Fan ArtHoe - - - 3.2 Static pressure test holes have been created, and test hole locations are well-marked and accessable. - - 3.2 A Massured returm setural static pressure: Plenum Cobinet Transition Other: - - 3.3 Measured returm setural static pressure: Plenum ICabinet Transition Other: - - 3.4 Measured static pressure: Plenum ICabinet Transition Other: - - - 3.4 Measured total externa	2.9 OEM subcooling goal: 'F DB		-				
2.11 Exaporator saturation temperature: IF DB (Using Item 2.5) - 2.12 Superheat value: IF DB (Item 2.6 - Item 2.11) - 2.13 OEM superheat goal: IF DB (Item 2.6 - Item 2.11) - 2.14 Superheat doublanc: IF DB (Item 2.6 - Item 2.11) - 2.14 Superheat doublanc: IF DB (Item 2.6 - Item 2.13) - 2.14 Superheat doublanc: IF DB (Item 2.12 - Item 2.13) - 2.15 Item 2.01 is a 3F or Item 2.14 is a 5FF - - 2.16 An DEM test procedure (e.g., as defined for a ground-source heat pump) has been used in place of the sub-cooling or super-heat process and documentation has been stached that defines this procedure - 3.1 Indoor HVAC Fan Arthew - - - 3.1 The mode with the higher design HVAC fan aitflow used, per Item 5.2 of National HVAC Design Report: - - Heating Diccooling - - - - 3.1 Massured return static pressure: Plenum Cabinet IT transition Other: - - - 3.3 Measured return static pressure: Plenum ICabinet IT ransition Other: - - - 3.3 Measured HVAC fan aitflow (Item 3.1 is at 15% of design HVAC fan aitflow testing); IWC - - 3.4 Measured supply extern	2.10 Subcooling deviation: *F DB (Item 2.8 - Item	2.9)	-				
2.12 Superheat value: T DB (herr 2.6 - herr 2.11) - 2.13 CM superheat goal: T DB (Living superheat tables and herrs 2.1 & 2.2) - 2.14 Superheat deviation: T DB (herr 2.6 - herr 2.11) - - 2.15 New 2.10 ha 3 DF or herr 2.14 is 3 DF - - - - 2.15 New 2.10 ha 3 DF or herr 2.14 is 3 DF -	For System with Fixed Orifice:						
2.13 OEM superheat goal: T DB (Using superheat tables and Barns 2.1 & 2.2) . 2.14 Superheat deviator: T DB (Isers 2.12 - Barn 2.13) . 2.15 Barn 2.10 is 3 2F or Barn 2.14 is a 5F . . 2.16 An DEM hast process and documentation has been attached that defines this procedure . . 2.16 An DEM hast process and documentation has been attached that defines this procedure . . 3.1 Indoor HVAC F an Aitline . . . 3.1 Indoor HVAC F an Aitline . . . 3.1 The mode with the higher design HVAC fan atflow used, per lisen 5.2 of National HVAC Design Report: . . 3.1 Black pressure leaf holes have been created, and leaf hole locations are well-marked and accessable. . . 3.2 Static pressure bash holes have been created, and leaf hole locations are well-marked and accessable. . . 3.4 Measured return static pressure (Enter value only, without pathway sign):	2.11 Evaporator saturation temperature: "# DB (Using Item 2.5)		-				
2.13 OEM superheat goal: T DB (Using superheat tables and Barns 2.1 & 2.2) . 2.14 Superheat deviator: T DB (Isers 2.12 - Barn 2.13) . 2.15 Barn 2.10 is 3 2F or Barn 2.14 is a 5F . . 2.16 An DEM hast process and documentation has been attached that defines this procedure . . 2.16 An DEM hast process and documentation has been attached that defines this procedure . . 3.1 Indoor HVAC F an Aitline . . . 3.1 Indoor HVAC F an Aitline . . . 3.1 The mode with the higher design HVAC fan atflow used, per lisen 5.2 of National HVAC Design Report: . . 3.1 Black pressure leaf holes have been created, and leaf hole locations are well-marked and accessable. . . 3.2 Static pressure bash holes have been created, and leaf hole locations are well-marked and accessable. . . 3.4 Measured return static pressure (Enter value only, without pathway sign):	2 12 Superheat value: # DB, (Rem 2.6 - Rem	210					
2.14 Superhead deviation: T DB (Item 2.12 – Item 2.13) - 2.15 Item 2.10 is a 37 F or Item 2.14 is a 57 F - - 2.16 An DEM test procedure (e.g., as defined for a ground-source heat pump) has been used in place of the sub-cooling or super-heat process and documentation has been attached that defines this procedure. - - 2.15 Interest MAC Fan ArtHour - - - - 3.1 Index PMAC Fan ArtHour - - - - 3.1 The mode with the higher design HMAC fan aitHour used, per Item 5.2 of National HVAC Design Report: - - - 11 Heating Cooling Cooling of Supply external static pressure: Planum [Cabinet [Transition]Other: - - 3.2 Static pressure learned static pressure: Planum [Cabinet [Transition]Other: - - - 3.3 Measured returns detarial static pressure: Planum [Cabinet [Transition]Other: - - - 3.4 Measured supply external static pressure: Planum [Cabinet [Transition]Other: - - - 3.5 Measured HWAC fan althour, using Item 3.5 and fan speed static pressure = [MWC - - - - 3.6 Measured HWAC fan althour, using Item 3.5 and fan speed static pressure = [MWC - -							
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Test hole location for supply external static pressure: Plenum @Cabinet Transition @Other: - - 3.3 Measured return static pressure [Chier value only, without regains sign):	_						
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3.5 Massured total external static pressure = Value-only from Item 3.3 + Value-only from Item 3.4 =							
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42 Room-by-room airflows verified by contractor to be within the greater of ± 20% or 25 CPM of design airflow		n National HVAC Design Report, and					
			_	-			

Page 2 of 6

Revised 09/01/2018

#3 - Streamlines ENERGY STAR Program Requirements

Streamlines ENERGY STAR program requirements

- An energy rating completed with certain features locked in:
 - Target score
 - Grade I insulation
 - Grade I or II HVAC grading
 - Minimum insulation levels, window/door ratings, duct leakage
- Plus:
 - 1. Bedroom pressure-balancing for comfort
 - 2. Reduced thermal bridging for comfort
 - 3. Air sealing details for efficiency and comfort
 - 4. Indoor air quality features for health
 - 5. Water management system features for durability, required by code

Status Update On HVAC Grading Standard



1. Standard 310: HVAC Grading Standard

 What it does: Defines how the Rater completes the design review, field tests, and designates the grade.

– Status:

- 1st comment period has concluded
- 2nd comment period should commence in November
- Aiming to finalize in Q1 2020

2. Standard 301: Energy Ratings Update (Non-calcs):

- What it does: Integrates Std. 310 into the overall rating process; updates definitions, minimum rated features, and on-site inspection protocols.
- Status:
 - Submitted in September
 - Aiming to finalize in Q1 2020



3. Std. 310 HVAC Design Report Templates:

 What it does: Incorporates Std. 310 design documentation requirements into Wrightsoft and RHVAC templates.

– Status:

- Discussions have started
- Aiming to finalize in Q2 2020



4. **RESNET Rater Training**:

- What it does: Trains raters on new requirements in Std. 310, prior to use.
- Status:
 - Development has started
 - Aiming to finalize in Q2 2020



5. Calculations Update:

- What it does: Updates standards and software to provide credit for properly installed HVAC systems.
- Status:
 - In process discussing with RESNET the value of rewarding properly installed HVAC systems in both ERI ratings and HERS ratings.
 - More to come..





ENERGY STAR Certified Homes

Web:

Home: www.energystar.gov/newhomespartners

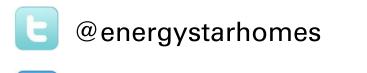
Technical: <u>www.energystar.gov/newhomesrequirements</u>

MESA: www.energystar.gov/mesa

Inbox Support energystarhomes@energystar.gov

Brice Lang U.S. EPA Partner Support Manager ENERGY STAR Certified Homes Lang.Brice@epa.gov

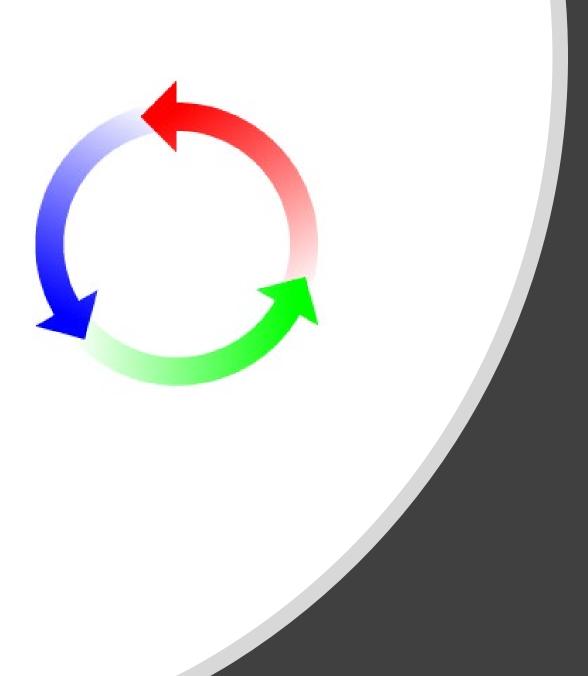
Social Media:



facebook.com/energystar

Dan Wildenhaus Rabble-Rouser BetterBuiltNW dwildenhaus@trccompanies.com





Where The Air Goes In, And Where The Comes Out: The importance of selecting Grilles Registers and Diffusers One to two feet in the High Velocity zone

Keep velocity below 50 fpm in occupied zone The Occupied Zone:

Two Zones:

The High Velocity Zone



Buoyancy: Warm air wants to rise, cold air wants to fall.

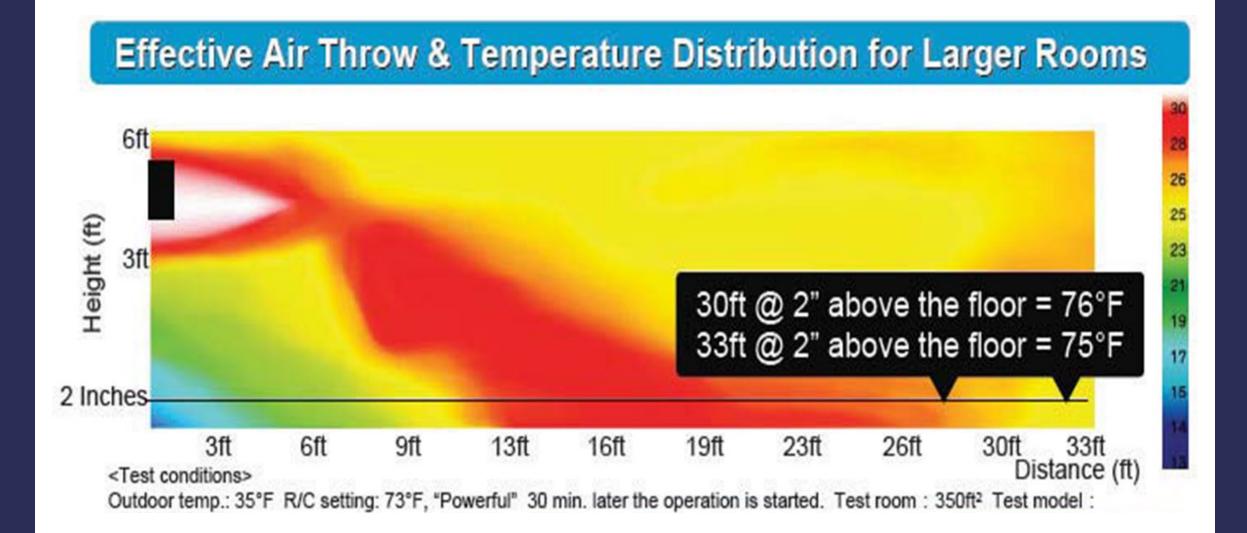
Heating Dominated: Floor

Cooling Dominated: Ceiling

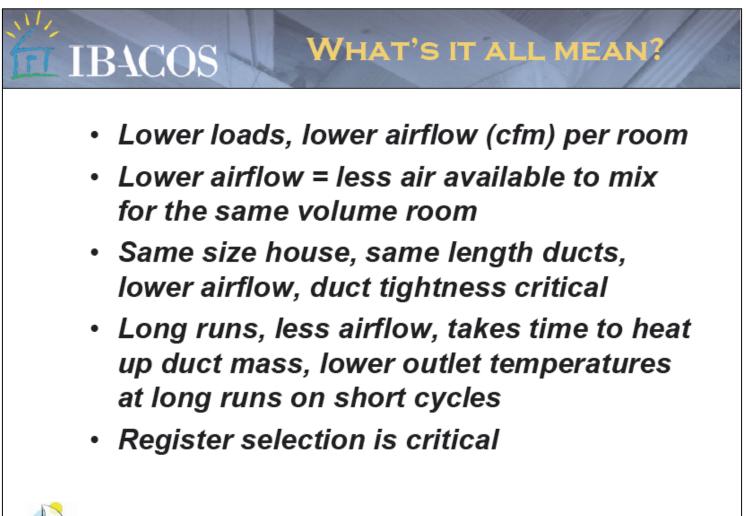
Real World Dominated: Cost

The Perfect Diffuser?





Its Even More Important in A Low Load House



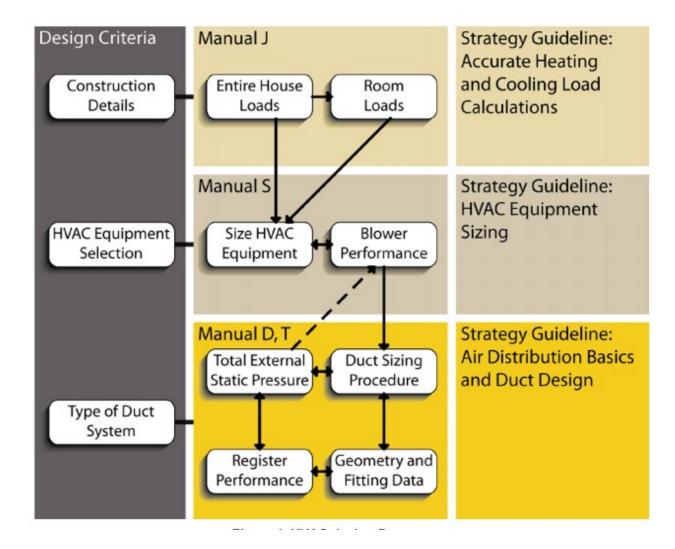




We Used To Be The Boss of Air!

	System size sf/ton	Air flow cfm/sf	Air exchange rate ACH nat
Historic "Rule of Thumb"	400	1.0	0.5 - 0.75
Energy Star – Cold Climate	1107	0.35	0.31
Energy Star – Mixed Humid Climate	1124	0.34	0.34
40% BA – Cold Climate	1476	0.26	0.10
40% BA – Mixed Humid Climate	1311	0.27	0.19

Courtesy of Ibacos





Before We Get Started:

Before you select and place a GRD you must:

- 1: Know the heat lost/gain per room
- 2: Know the cfm per room
- 3: Size the duct system correctly
- 4: Install the duct work correctly
- 5: Eat your vegetables and floss daily

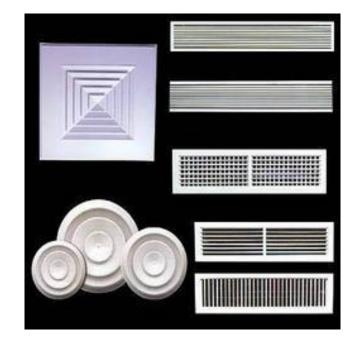
A Few Definitions

Grille: A louvered covering for an opening through which air passes.

Register: A grille which is equipped with a damper or control valve, and which directs air in a nonspreading jet.

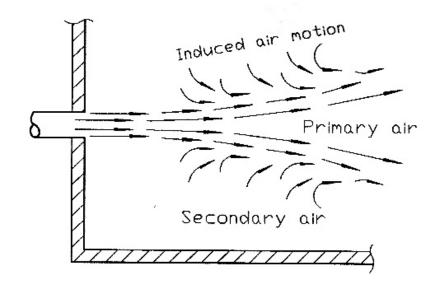
Diffuser An outlet discharging supply air in a spreading pattern.

Stratification Boundary The boundary between room air currents



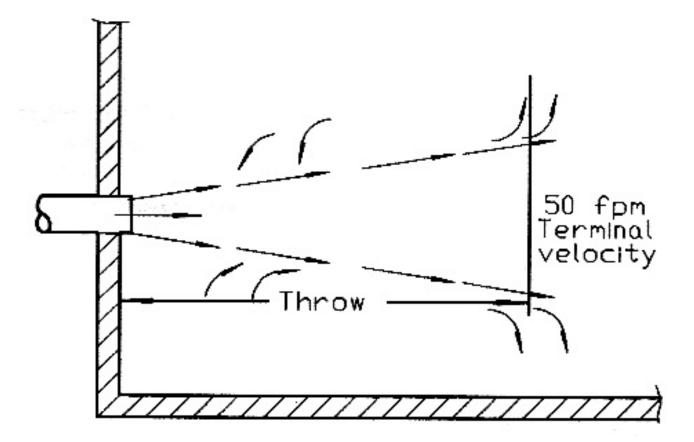
Air Mixing in a Zone. It's all about entrainment

- Primary air induces or set in motion the air surrounding it.
- Secondary air is 10 to 20 times the volume of the primary air.
- Hence 80 cfm of primary air sets 800 to 1600 cfm of secondary air into motion.



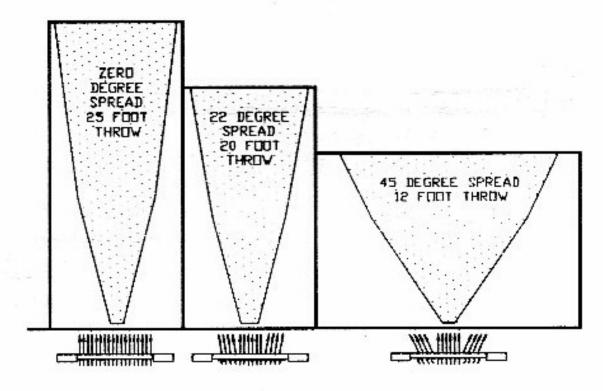
Register Throw

 Throw is the distance the air travels before it reaches some specific value (usually 50 feet per minute).

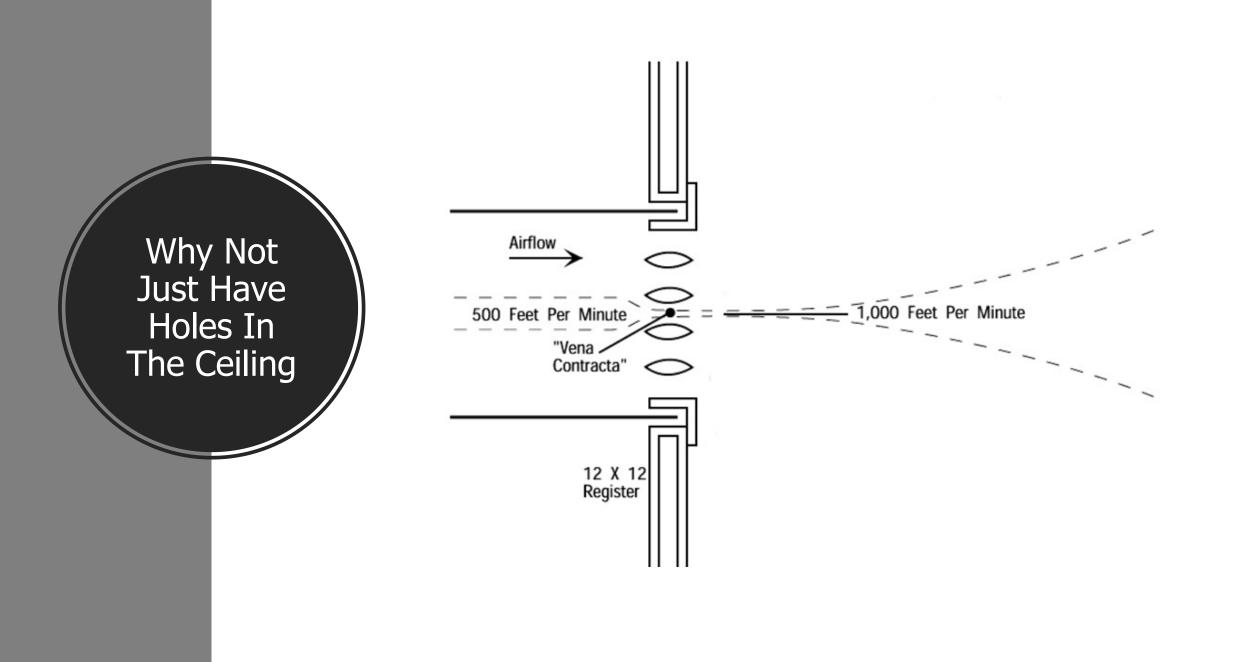


Register Spread

- Spread refers to how wide the jet of primary becomes
- Spread decreases and throw increases



THROW DEPENDS ON SPREAD

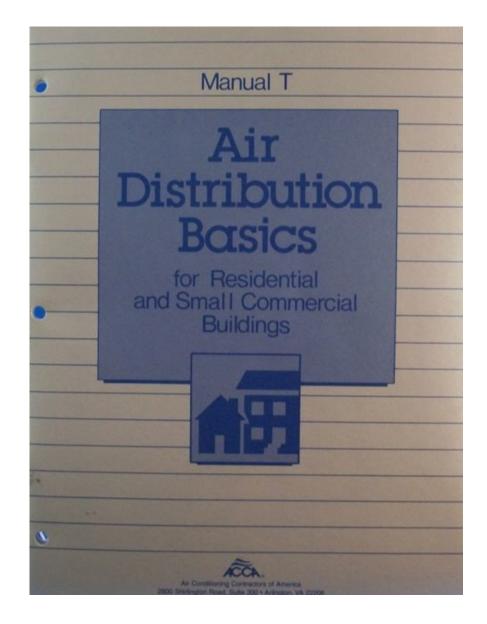


Grille and Register Selection

• ACCA's Manual "T" gives selection criteria

Good selection:

- Provides good mixing
- Prevents "cold drafts"
- Reduces noise
- Increases aesthetic appeal
- Good registers can increase cost \$20 to \$40 per grille/register.



Think High Side Wall Delivery

- Helps To Make the Duct System Compact
- Keeps higher velocity air out of the comfort zone
- Performs well in heating and cooling
- Uses outside wall and window to "drop" the air

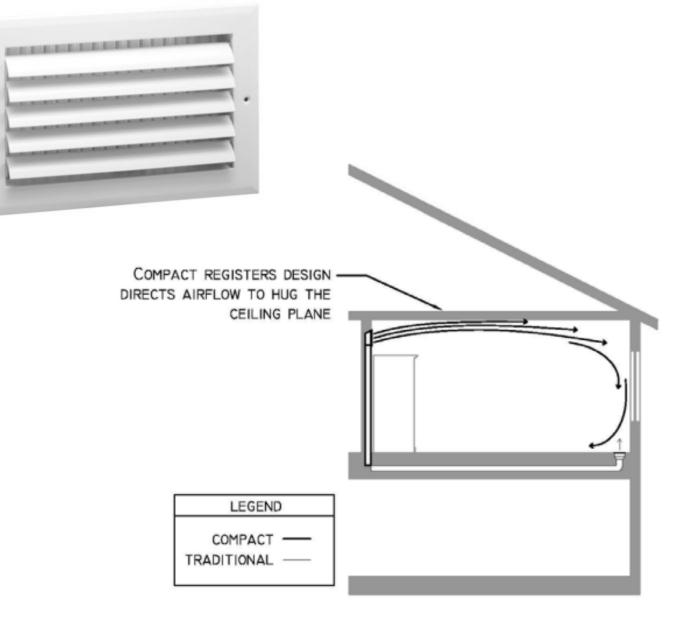


Figure 4. High sidewall versus floor register—section view

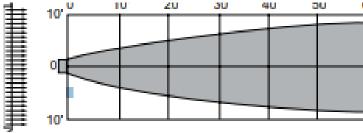
What's the Best High Side Wall Register size for 150CFM In a room that's 15ft long

Face Velo	city	300	400	500	600	700	800	900	1000	1100	1200	
Pressure		.006	.010	.016	.022	.031	.040	.050	.062	.075	.090	
8 x 4	CFM	45	60	80	95	110	125	140	155	170	185	
Ak .156	Throw	5.0	6.5	8.5	10.0	12.0	13.0	15.0	16.0	18.0	19.0	
10 x 4	CFM	60	80	100	120	140	160	180	200	220	240	
	Throw	6.0	7.5	9.5	12.0	13.0	15.0	17.0	19.0	20.0	22.0	
12 x 4	CFM	70	95	120	145	170	190	215	240	265	290	
Ak .240	Throw	6.0	8.0	10.0	12.0	14.0	16.0	18.0	20.0	22.0	25.0	
10 x 6	CFM	95	125	155	190	220	250	280	315	345	375	
Ak .313	Throw	7.0	9.0	12.0	14.0	16.0	19.0	21.0	23.0	26.0	28.0	
	CFM	115	150	190	225	265	305	340	380	415	455	
Ak .379	Throw	8.0	10.0	13.0	15.0	18.0	21.0	23.0	26.0	28.0	31.0	
	CFM	130	170	215	255	300	340	385	425	470	510	
Ak .425	Throw	8.0	11.0	14.0	16.0	19.0	21.0	24.0	27.0	30.0	32.0	
14 x 6	CFM	135	180	225	270	310	355	400	445	490	54	
Ak .446	Throw	8.0	11.0	14.0	17.0	19.0	22.0	25.0	28.0	30.0	33.0	
12 x 8	CFM	160	200	265	320	370	425	475	530	585	635	
Ak .530	Throw	9.0	11.0	15.0	18.0	21.0	24.0	27.0	30.0	33.0	36.0	
14 x 8	CFM	185	250	310	370	435	495	560	620	680	745	
Ak .620	Throw	10.0	13.0	16.0	20.0	23.0	26.0	30.0	33.0	36.0	39.0	
	For sizes not listed or sizing tips see page 35											
Terminal Ve	Terminal Velocity of 75 FPM											

A618 Adjustable Fin Register

(Page 17)





Pick a ceiling register for 80 CFM 80 CFM

-

A618MS/A618OB Register

- · Aluminum face/fin construction
- · Adjustable straight fins provides individual deflection control
- · Stamped face margins eliminate mitered corners and rough edge
- · Galvanized multi-shutter valve or

Face Velocity		300	400	500	600	700	800	900	1000
Pressure	Pressure Loss		.010	.016	.022	.031	.040	.050	.062
8 x 6	CFM	40	55	70	90	100	110	125	140
Ak .140	Throw	5.0	6.0	8.0	9.0	11.0	12.0	14.0	15.0
10 x 6	CFM	55	80	95	110	130	150	165	185
Ak .185	Throw	5.0	7.0	/10.0	11.0	13.0	14.0	16.0	18.0
12 x 6	CFM	70	90	115	135	160	180	205	225
Ak .225	Throw	6.0	8.0	12.0	12.0	14.0	16.0	18.0	20.0
14 x 6	CFM	85	115	145	175	205	230	260	290
Ak .290	Throw	7.0	9.0	16.0	14.0	17.0	19.0	21.0	24.0
16 x 6	CFM	100	130	165	200	230	265	295	330
Ak .330	Throw	8.0	10.0	18.0	15.0	18.0	20.0	23.0	25.0
18 x 6	CFM	115	155	195	235	275	310	350	390
Ak .390	Throw	8.0	11.0	21.0	17.0	20.0	22.0	25.0	28.0
20 x 6	CFM	130	175	220	265	310	360	395	440
Ak .440	Throw	5.0	12.0	24.0	18.0	21.0	24.0	27.0	30.0

Face Velocity 300 500 600 700 1000 400 800 900 CFM 28 38 47 57 66 76 85 95 8×4 .02 .02 Ps. .02 .03 .04 .05 .06 .07 Ak .095 Throw 1.5 2.0 2.5 4.5 3.0 3.5 4.0 5.0 59 CFM 35 47 7Ø 82 94 105 117 10 x 4 .d2 .02 .01 .01 .05 .06 Ps. .03 04Ak .117 2.5 4.5 1.5 2.0 3.0 3.5 4.0 5.0 Throw CFM 58, 97 135 154 174 193 77 116 10×6 .oh 02 .02 .05 .06 .01 .03 .04 Ak.,193 Throw 1.5 2.0 2.5 3.0 3.5 4.5 5.0 5.5 95 119 167 CFM 71 143 190 214 238 12 x 6 .01 .01 .01 .02 .04 .05 .06 .03 Ps. Ak .238 5.5 Throw 7.0 9.0 10.5 12.5 14.0 16.0 17.5 CFM 116 145 174 204 233 262 291 87 14 x 6 .01 .02 .02 .04 .05 .06 Ps. .01 .03 Ak .291 NA NA NA NA NA NA NA Throw NA 158 198 237 277 356 395 CFM 316 119 14 x 8 .02 .05 .01 .01. .01 .03 .04 .06 Ps. Ak .395 Throw NA NA NA NA NA NA NA NA Terminal Velocity of 75 FPM NA = Not Available

Terminal Velocity of 75 FPM



421 Floor Diffuser

- All-steel construction
- Multi-angle fin setting
- Rolled fin for strength and safety
- Welded construction
- Foot-operated dial control
- Heavy-gauge stamped face
- · Golden Sand or Bright White enamel finish

420/421 Floor Diffuser (Page 7)

Face Velo	300	400	500	600	700	800	900	1000	
Pressure L	.088	.006	.010	.016	.022	.031	.040	.050	.062
2 x 10	CFM		35	45	50	60	70	75	85
Ak .085	Spread		3.0	5.0	5.0	6.0	7.0	8.0	9.0
AK .000	Throw		4.0	4.5	6.0	(7.0	8.0	.050 75	10.0
2 x 12	CFM	- 30	40	50	60	70	80	90	100
Ak .100	Spread	3.0	4.0	4.5	5.5	6.5	7.0	8.0	9.0
AK.100	Throw	3.5	4.5	5.5	7.0	8.0	9.0	.0 8.0 .0 9.0 .0 9.0 .0 8.0 .0 10.0 .0 10.5 .0 9.0 .0 10.5 .0 9.0 .5 10.5 .5 9.5 .0 11.0	11.0
2 x 14	CFM	35	45	60	70	80	90	105	115
Ak .115	Spread	3.5	4.0	<u>-5.Q</u>	7.0	7.0	8.0	9.0	10.0
AK.IIS	Throw	3.5	4.5	6.0	8.0	8.0	9.5	0 .050 0 75 0 8.0 0 9.0 0 9.0 0 9.0 0 9.0 0 9.0 0 10.0 0 10.5 5 10.5 5 9.5 0 11.0 5 155 0 11.5	12.0
4 x 8	CFM	40	50	65	80	90	105	115	130
Ak .130	Spread	3.0	4.8	-5,0	6.5	7.5	8.5	9.5	11.0
AK . 130	Throw	4.0	4.5	6,0	7.5	8.5	10.0	11.0	13.0
4 x 10	CFM	50	70	85	100	120	135	155	170
	Spread	4.5	5.0	6.5	7.5	9.0	10.0	11.5	13.0
Ak .170	Throw	4.0	6.0	8.0	10.0	11.0	12.5	8.0 9 9.0 10 9.0 10 9.0 10 10.0 11 10.0 11 10.0 11 10.0 11 10.5 12 5 11.5 5 15.5 11.0 13 5 15.5 11.5 13	15.5



Rezzin Plastic Floor Diffuser

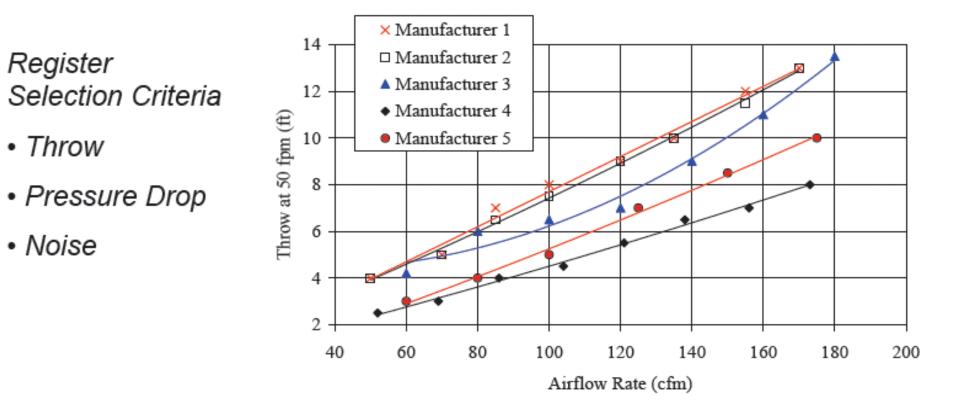
- Solid ABS construction for floor use
- Multi-angled fins for fan pattern of air distribution
- · Foot-operated, opposed-blade damper
- Bright White or Brown finish

Rezzin Floor Diffuser (Page 8)

Face Velocity		300	400	500	600	700	800	900	1000
	CFM	25	- 34	42	50	59	67	76	84
2 x 12	Ps	.01	.02	.03	.05	.06	.08	.10	.12
Ak .084	Throw	2.0	2.5	3.5	4.0	(4.5) 5.5	6.0	6.5
	Spread	1.5	2.0	2.5	3.0	3.0	3.5	4.0	4.5
	CFM	42	56	71	85	99	113	127	141
4 x 10	Ps	.02	.02	.03	.04	.06	.07	.09	.11
Ak .141	Throw	2.0	2.5	3.0	3.5	4.5	5.0	5.5	6.0
	Spread	0.5	1.5	2.5	3.0	4.0	5.0	5.5	6.5
	CFM	47	63	79	94	110	126	141	157
4 x 12	Ps	.02	.63	.04	.05	.07	.09	.11	.13
Ak .157	Throw	1.5	2.0	2.5	3.0	3.5	4.0	4.5	5.0
	Spread	0.5	1.5	2.5	4.0	5.0	6.0	7.0	8.0

Terminal Velocity of 50 FPM

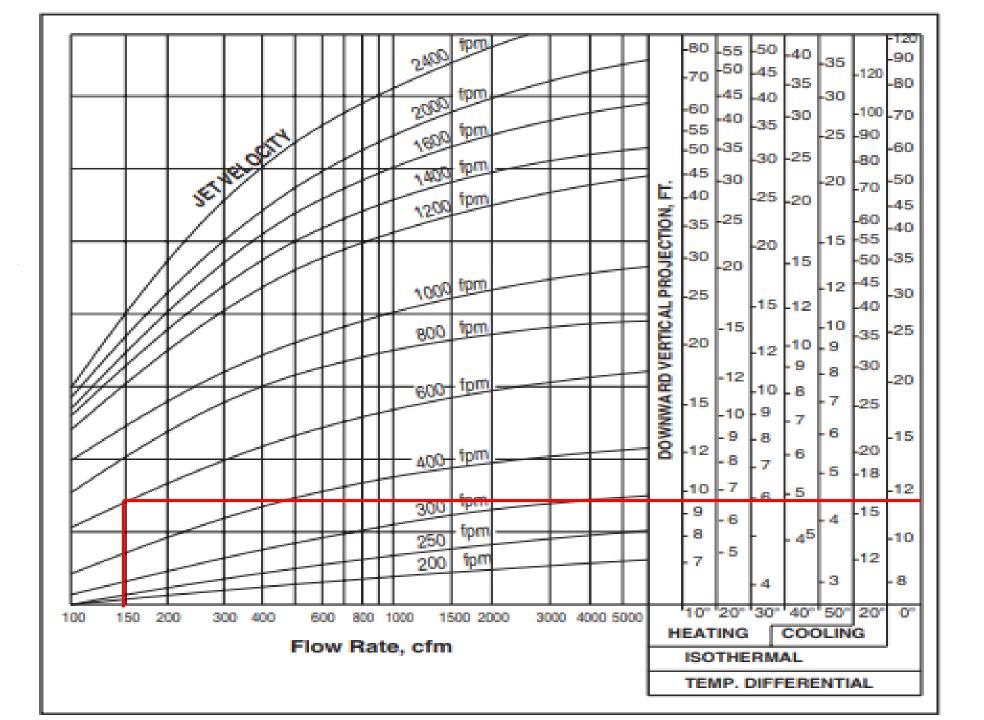
Performance of Various Supply Registers



Floor Register Performance (10x4)

Courtesy of Ibacos

Isothermal Correction Or How to Cut the downward throw in half



One More Vegetable: Entry Conditions Matter

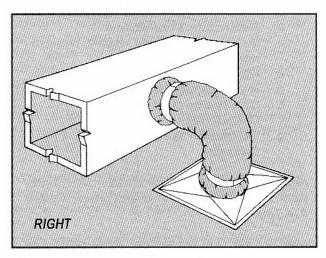
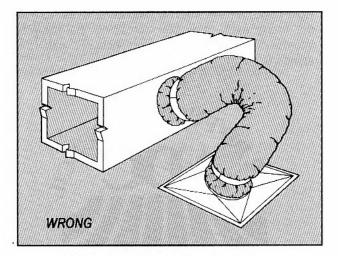
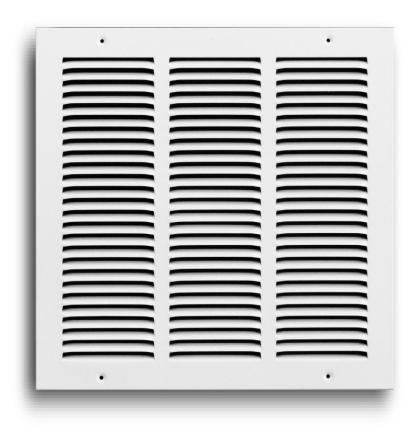


Figure 6





Return: Just a hole for the air to get back to the air handler



- They have very little impact on air distribution patterns
- Bigger is better
 - Better filtration
 - Lower Noise
 - More static (muscle) for the supply side

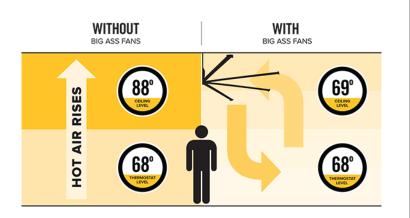
Sizing Returns

- 400 500m fpm max
- 300 max for filter grilles
- On stamp faced grilles, keep it below 400 fpm for noise purposes
- Rule of thumb 2 cm per filter size

Filter	Size	Area (in²)	Ton (cfm)	Filte	r Size	Area (in²)	То
12	12	144	n/a	20	20	400	2
12	20	240	1 (400)	20	25	500	2.5
12	- 24	288	1.5 (600)	20	30	600	3
12	30	360	1.5 (600)	20	36	720	3
14	14	196	1 (400)	24	24	576	3
14	20	280	1.5 (600)	- 24	30	720	3
14	24	336	1.5 (600)	- 24	36	864	4
14	30	420	2 (800)	25	25	625	3
16	20	320	1.5 (600)	- 30	30	900	4
16	- 24	384	2 (800)	30	36	1080	5

A Few Guidelines









What They Do in the South:

Zen Like Question For The Day

 How should you
 A: Don't worry about it design ducts and
 B: At the top speed select registers for
 C: At the speed it will be a variable speed system?